

Derivation of 2-D Thermoelastic Equations of Motion for a Finite Cylinder

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Outline

- Previous work by Wimett
- Current Work
- Results and Graphs
- Future Work

Why Do We Care?

- Thermoelastic equations can give accurate results when modeling material movement from forces due to temperature changes
 - Stresses, strains, displacements can be found
 - Applicable to criticality accident analysis
- Difficulties arise from modeling complicated geometries
 - “Easiest” is 1-D spherical geometry
 - “Second easiest” is simple 2-D cylindrical geometry
 - Godiva IV, SPR II can provide good experimental data to test against

Previous Work

- Wimettt (1992) modeled the radial direction only, using approximations for the axial direction
 - Disk Approximation
 - Cylinder Approximation
 - Analytical solutions were found and compared to SPR II data
- Myers, et al (1995) developed original MRKJ, using a coupled neutronic-hydrodynamic method
- MRKJ was modified to use coupled neutronic-thermoelastic method
 - Spherical geometry only

Current Work

- Goal: Apply the neutronic-thermoelastic method to a 2-D cylindrical system
 - Derive the thermoelastic equations in 2-D cylindrical geometry
 - Test equations for simple problems
 - Implement new model into neutronic-thermoelastic code
 - Compare results to experimental data
- First two have been completed, currently working on part 3

Derivation

- Start from three sets of equations:
 - Stress-Strain Relations
 - Equations of Motion in Cylindrical Geometry
 - Strain-Displacement Relations in Cylindrical Geometry
- Concerned with r, z directions, so set derivatives with respect to θ equal to zero and θ displacements equal to zero
- Combine equations for final equation

Derivation: Stress-Strain Relations

$$\varepsilon_{rr} = \frac{1}{E} [\sigma_{rr} - \nu(\sigma_{\theta\theta} + \sigma_{zz})] + \alpha\Delta T$$

$$\varepsilon_{\theta\theta} = \frac{1}{E} [\sigma_{\theta\theta} - \nu(\sigma_{rr} + \sigma_{zz})] + \alpha\Delta T$$

$$\varepsilon_{zz} = \frac{1}{E} [\sigma_{zz} - \nu(\sigma_{rr} + \sigma_{\theta\theta})] + \alpha\Delta T$$

$$\sigma_{r\theta} = 0$$

$$\sigma_{z\theta} = 0$$

$$\varepsilon_{zr} = \frac{1}{2G} \sigma_{zr}$$

Derivation: Equations of Motion

$$\frac{\partial \sigma_{rr}}{\partial r} + \frac{\partial \sigma_{rz}}{\partial z} + \frac{\sigma_{rr} - \sigma_{\theta\theta}}{r} = \rho \frac{\partial^2 u}{\partial t^2}$$

$$\frac{\partial \sigma_{rz}}{\partial r} + \frac{\partial \sigma_{zz}}{\partial z} + \frac{\sigma_{rz}}{r} = \rho \frac{\partial^2 w}{\partial t^2}$$

Derivation: Strain-Displacement Relations

$$\varepsilon_{rr} = \frac{\partial u}{\partial r}$$

$$\varepsilon_{\theta\theta} = \frac{u}{r}$$

$$\varepsilon_{zz} = \frac{\partial w}{\partial z}$$

$$\varepsilon_{r\theta} = 0$$

$$\varepsilon_{rz} = \frac{1}{2} \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial r} \right)$$

$$\varepsilon_{z\theta} = 0$$

Derivation: Final Equations

$$\rho \frac{\partial^2 u}{\partial t^2} = \frac{E(\nu-1)}{(2\nu-1)(\nu+1)} \left[\frac{\partial^2 u}{\partial r^2} + \frac{1}{r} \frac{\partial u}{\partial r} - \frac{u}{r^2} \right] + \frac{E}{2(1+\nu)} \frac{\partial^2 u}{\partial z^2} - \frac{E}{2(2\nu-1)(\nu+1)} \frac{\partial^2 w}{\partial z \partial r} + \frac{E}{2\nu-1} \alpha \frac{\partial \Delta T}{\partial r}$$

$$\rho \frac{\partial^2 w}{\partial t^2} = \frac{E(\nu-1)}{(2\nu-1)(\nu+1)} \frac{\partial^2 w}{\partial z^2} + \frac{E}{2(1+\nu)} \left[\frac{\partial^2 w}{\partial r^2} + \frac{1}{r} \frac{\partial w}{\partial r} \right] - \frac{E}{2(2\nu-1)(\nu+1)} \left[\frac{\partial^2 u}{\partial z \partial r} + \frac{1}{r} \frac{\partial u}{\partial z} \right] + \frac{E}{2\nu-1} \alpha \frac{\partial \Delta T}{\partial z}$$

Derivation: Boundary and Initial Conditions

- Left Boundary Conditions:

$$u(r = 0) = 0$$

$$w(z = 0) = 0$$

$$\frac{\partial u(z = 0)}{\partial z} = 0$$

$$\frac{\partial w(r = 0)}{\partial r} = 0$$

$$\sigma_{rr}(r = R_i) = 0$$

$$w(z = 0) = 0$$

$$\frac{\partial u(z = 0)}{\partial z} = 0$$

$$\sigma_{rz}(r = R_i) = 0$$

- Outer Boundary Conditions (zero stress condition):

$$\sigma_{rr}(r = R) = 0$$

$$\sigma_{zz}(z = H) = 0$$

$$\sigma_{rz}(r = R) = 0$$

$$\sigma_{rz}(z = H) = 0$$

- Initial Conditions (at rest):

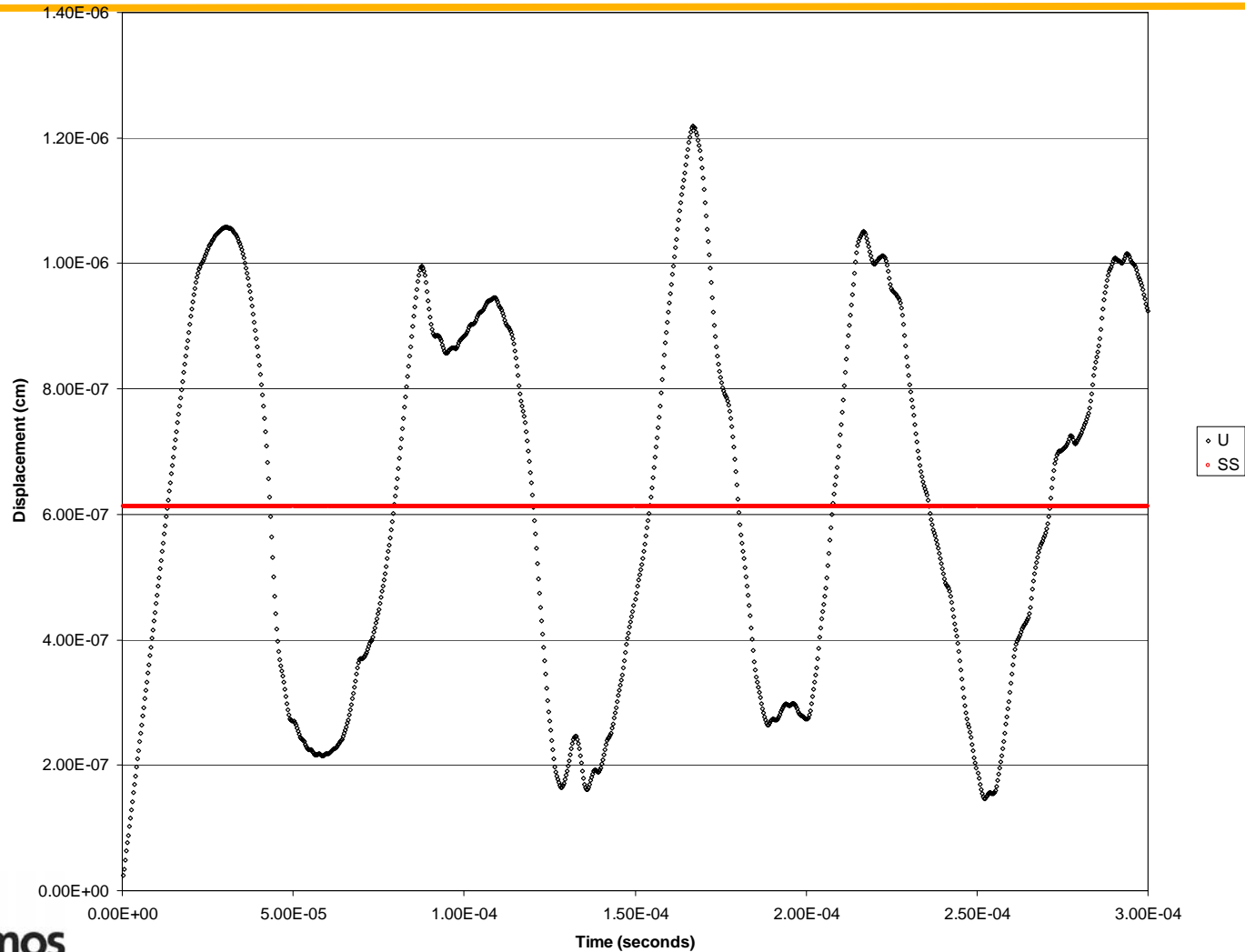
$$u(t = 0) = 0 \quad \frac{\partial u(t = 0)}{\partial t} = 0$$

$$w(t = 0) = 0 \quad \frac{\partial w(t = 0)}{\partial t} = 0$$

Results

- Small cylinder of Uranium
 - Diameter: 6"
 - Height: 6"
 - Constant Temperature Difference: 1 degree Celsius
- Boundary Conditions
 - Outer: Zero outer stresses
 - Inner: Zero displacements at origin
- Finite Difference Algorithm
 - Explicit Scheme
- Output
 - Displacements
 - Stresses, Strains

Displacement



Future Work

- Use this model in cylindrical MRKJ
 - Godiva IV, SPR
- Complications
 - No longer can use 1-D Partisn code without additional adjustments
 - Godiva IV has inner and outer core